



Dynamic Stresses Near the Working Surface from A Plane Wave

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ABSTRACT

The paper considers the reaction of a cylindrical layer located in a viscoelastic medium, when exposed to a transverse (longitudinal) seismic wave. It has been established that the maximum dynamic stress is 10-15% higher than the static one, and the wave numbers at which the maximum value is reached.

Keywords:

spherical shell; spherical protective domes; hesitation; cubic equation

We assume that the harmonic wave is plane and that the wave front is parallel to the axis of the cylindrical layer (Fig. 1). The basic

equations of the theory of viscoelasticity for this plane strain problem are reduced to the following equation

$$\Delta \varphi - \int_{-\infty}^t [R_{\lambda}(t-\tau) + 2R_{\mu}(t-\tau)] \Delta \varphi d\tau = \frac{1}{a^2} \frac{\partial^2 \varphi}{\partial t^2}; \quad \Delta \vec{\psi} - \int_{-\infty}^t R_{\mu}(t-\tau) \Delta \vec{\psi} d\tau = \frac{1}{b^2} \frac{\partial^2 \vec{\psi}}{\partial t^2} \quad (1)$$

where $\alpha^2 = (\lambda_0 + 2\mu_0)/\rho$; $b^2 = \mu_0/\rho$; φ и ψ – movement potentials. $R_{\lambda}(t-\tau)$ and $R_{\mu}(t-\tau)$ – relaxation core; ν - Poisson's ratio, which we consider to be a non-relaxing quantity [1]. The

incident plane wave is considered to propagate in the positive direction of the x-axis and is represented as

$$\text{follows: } \begin{cases} \varphi^{(i)} = \varphi_0 e^{i(\alpha x - \omega t)}, \varphi^{(i)} = 0 - \text{when exposed to prolonged pain} \\ \psi^{(i)} = \psi_0 e^{i(\beta x - \omega t)}, \psi^{(i)} = 0 - \text{when exposed to shear waves} \end{cases} \quad (2)$$

φ_0 и ψ_0 – amplitude values; ω - circular frequency; α and β are wave numbers, which must be complex numbers $\alpha = \alpha_R + i\alpha_I$; $\beta = \beta_R + i\beta_I$ $\alpha_I < 0$ and $\beta_I < 0$ denote the attenuation

coefficients; α_R and β_R denote the wave numbers of longitudinal waves and shear waves, respectively. The solution of equation (1) can be sought in the form:

$$\varphi(r, \theta, t) = \sum_{k=1}^{\infty} \varphi_k(r, \theta) e^{i\omega t}; \quad \psi(r, \theta, t) = \sum_{k=1}^{\infty} \psi_k(r, \theta) e^{i\omega t}, \quad (3)$$

где $\varphi_k(r, \theta)$ и $\psi_k(r, \theta)$ – real functions satisfying the equations

$$\Delta \Phi_k + \frac{\alpha_k}{1-L_k} \Phi_k = 0; \Delta \psi_k + \frac{\beta_k}{1-M_k} \psi_k = 0; \quad (4)$$

$$L_k = \int_0^\infty [R_\lambda(\xi) + 2R_\mu(\xi)] \exp(-i\omega\xi) d\xi, M_k = \int_0^\infty R_\mu(\xi) \exp(-i\omega\xi) d\xi.$$

To describe the viscoelastic properties of the material, the Boltzmann-Volterra theory with the Rzhnitsyn-Koltunov relaxation kernel in the form $R(t) = Ae^{-\beta t} t^{\alpha-1}$. In this case, the sine $\Gamma(s)$ and cosine $\Gamma(c)$ of the Fourier samples, the relaxation kernel $R(t)$ is expressed in terms of the $\Gamma(\alpha)$ -Gamma function

$$\Gamma^s = \frac{A\Gamma(\alpha)}{(\omega^2 + \beta^2)^{\alpha/2}} \sin(\alpha \arctg \frac{\omega}{\beta}), \quad \Gamma^c = \frac{A\Gamma(\alpha)}{(\omega^2 + \beta^2)^{\alpha/2}} \cos(\alpha \arctg \frac{\omega}{\beta}).$$

The solution of equation (4) is expressed in terms of the Hankel functions of the 1st and 2nd kind of

$$\varphi = \sum_{n=0}^{\infty} [A_n H_n^{(1)}(\alpha^* r) + A_n' H_n^{(2)}(\alpha^* r)] \cos n\theta e^{-i\omega t}$$

the nth order:

$$\psi = \sum_{n=0}^{\infty} [B_n H_n^{(1)}(\beta^* r) + B_n' H_n^{(2)}(\beta^* r)] \sin n\theta e^{-i\omega t}, \quad (5)$$

$$\alpha_{k1}^{*2} = \frac{\alpha_{k1}}{1-L_k}; \beta_k^{*2} = \frac{\beta_k}{1-M_k},$$

where A_n, A_n', B_n, B_n' – expansion coefficients, which are determined by the corresponding boundary conditions; $H_n^{(1)}(\alpha^* r)$ and $H_n^{(2)}(\alpha^* r)$ – Hankel function of the 1st and 2nd kind of $r \rightarrow \infty$ to the Sommerfeld radiation condition [1]:

the nth order, respectively

$H_n^{(2)}(\alpha r) = I_n(\alpha r) - iN_n(\alpha r)$. Solution (5) satisfies at infinity

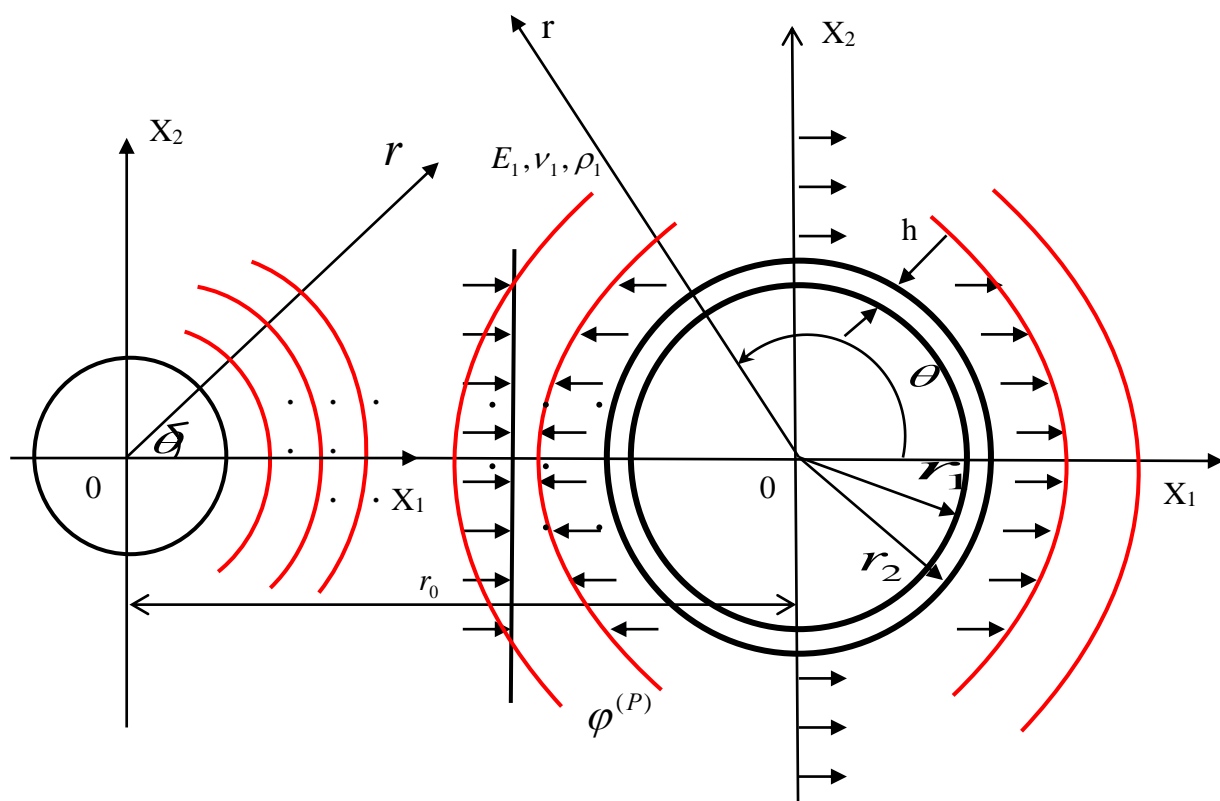


Fig.1. Calculation scheme of a cylindrical reinforced hole in a viscoelastic medium

$$\lim_{r \rightarrow \infty} \varphi = 0, \quad \lim_{r \rightarrow \infty} (\sqrt{r})^\kappa \left(\frac{\partial \varphi}{\partial r} + i\alpha \varphi \right) = 0, \\ \lim_{r \rightarrow \infty} \psi = 0, \quad \lim_{r \rightarrow \infty} (\sqrt{r})^\kappa \left(\frac{\partial \psi}{\partial r} + i\beta \psi \right) = 0.$$

For this, $A'_n = B'_n = 0$ must be present. The solution of equation (5) is represented as:

$$\varphi^{(r)} = \sum_{n=0}^{\infty} A_n H_n^{(1)}(\alpha^* r) \cos n\theta e^{-i\omega t}; \quad (6) \\ \psi^{(r)} = \sum_{n=0}^{\infty} B_n H_n^{(1)}(\beta^* r) \sin(n\theta) e^{-i\omega t}.$$

The total potential can be determined by superimposing the potentials of the incident and reflected waves. Thus, the displacement potentials will be [1]:

$$\varphi = \varphi^{(i)} + \varphi^{(k)}, \varphi^{(p)} = \varphi(r, \theta, t), \psi = \psi^{(i)} + \psi^{(k)}, \psi^{(p)} = \psi(r, \theta, t) \quad (7)$$

It follows that stresses and displacements can easily be expressed in terms of displacement potentials

$$u_r = \frac{\partial \varphi}{\partial r} + \frac{1}{r} \frac{\partial \psi}{\partial \theta}; \quad u_\theta = \frac{1}{r} \frac{\partial \varphi}{\partial \theta} - \frac{\partial \psi}{\partial r}, \\ \varepsilon_{rr} = \frac{\partial u_r}{\partial r}; \quad \varepsilon_{\theta\theta} = \frac{1}{r} \frac{\partial u_\theta}{\partial \theta} + \frac{u_r}{r}; \quad \varepsilon_{r\theta} = \frac{1}{2} \left(\frac{1}{r} \frac{\partial u_r}{\partial \theta} + \frac{\partial u_\theta}{\partial r} + \frac{u_\theta}{r} \right), \\ \sigma_{rr} = \tilde{\lambda} \nabla^2 \varphi + 2\tilde{\mu} \left[\frac{\partial^2 \varphi}{\partial r^2} + \frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial \varphi}{\partial \theta} \right) \right]; \\ \sigma_{\theta\theta} = \tilde{\lambda} \nabla^2 \varphi + 2\tilde{\mu} \left[\frac{1}{r} \left(\frac{\partial \varphi}{\partial r} + \frac{1}{r} \frac{\partial^2 \varphi}{\partial \theta^2} \right) + \frac{1}{r} \left(\frac{1}{r} \frac{\partial \psi}{\partial \theta} - \frac{\partial^2 \psi}{\partial r \partial \theta} \right) \right]; \quad (8) \\ \sigma_{zz} = \tilde{\lambda} \nabla^2 \varphi; \quad \sigma_{r\theta} = 2\tilde{\mu} 2 \left(\frac{1}{r} \frac{\partial^2 \varphi}{\partial \theta \partial r} - \frac{1}{r^2} \frac{\partial \varphi}{\partial \theta} \right),$$

$$\text{where} \quad \{(\tilde{\lambda}, \tilde{\mu}) f(t)\} = (\lambda_0, \mu_0) \left[f(t) - \int_{-\infty}^t R_{\lambda, \mu}(t - \tau) f(\tau) d\tau \right],$$

$f(t)$ – some function; u_r – radial displacement; u_θ is the tangential displacement; $\varepsilon_{rr}, \varepsilon_{\theta\theta}, \varepsilon_{r\theta}$ – strain tensor elements; $\sigma_{rr}, \sigma_{r\theta}, \sigma_{\theta\theta}, \sigma_{zz}$ – stress tensor elements. As mentioned above, the coefficients A_n and B_n are determined from the corresponding boundary conditions.

The boundary conditions for $r = a$, and is the radius of the cylindrical discontinuity surface will be: $\sigma_{rr} = 0$; $\sigma_{r\theta} = 0$.

The coefficients A_n and B_n are determined from the corresponding boundary conditions for each value of n . Thus, the stress concentration in the flow under the influence of a shear wave (2) takes the following value[1]

$$\sigma_{\theta\theta}^* = -\frac{8}{\pi} \left(1 - \frac{1}{K_*^2}\right) \sum_{n=1}^{\infty} i^{n+1} \frac{n(n^2 - 1 - \frac{\Omega_2}{2}) H_n(\Omega_1)}{\delta_n} \sin n\theta;$$

$$\delta_n = \Omega_2^2 (n^2 + n - \frac{\Omega_2^2}{4}) H_n(\Omega_1) H_n(\Omega_2) + \Omega_1 \Omega_2 (n^2 - 1) H_{n-1}(\Omega_1) H_{n-1}(\Omega_2) +$$

$$+ (n^2 - n^2 - \frac{\Omega_2^2}{2}) [\Omega_2 H_n(\Omega_1) H_{n-1}(\Omega_2) + \Omega_1 H_{n-1}(\Omega_1) H_n(\Omega_2)];$$

$$\Omega_1 = \alpha_1 a \quad ; \quad \Omega_2 = \beta_1 a \quad ; \quad K_*^2 = \frac{\beta_*^2}{\alpha_*^2} = \frac{C_{\alpha_*}''}{C_{\beta_*}^2} = \frac{2(1-\nu)\Gamma_1^*}{(1-2\nu)\Gamma_2^*}$$

Γ_1^* и Γ_2^* - describe the viscoelastic properties of the material.

In the case of an elastic cylindrical body in a viscoelastic medium, rigid contact conditions are set at the boundary $r = a$, under which stresses and displacements are continuous at the boundary:

$$\sigma_{rr1} = \sigma_{rr2} \quad ; \quad \sigma_{r\theta1} = \sigma_{r\theta2};$$

$$u_{r1} = u_{r2} \quad ; \quad u_{\theta1} = u_{\theta2}$$

Where, σ_{rr1} , $\sigma_{r\theta1}$ - stresses in a viscoelastic environment; σ_{rr2} and

$\sigma_{r\theta2}$ - viscoelastic inclusion stresses; u_{r1} , $u_{\theta1}$ - radial and tangential displacements of the environment; u_{r2} , $u_{\theta2}$ - radial and tangential displacements of the elastic inclusion. If there is no friction at the contact boundary, then $\sigma_{r\theta1} = \sigma_{r\theta2} = 0$; $u_{r1} = u_{r2}$.

Determination of stresses at $r = a$. Near the cylindrical cavity, the contour stresses $\sigma_{\theta\theta}$ at $r = a$ express the stress concentration. The contour stress under the influence of a longitudinal harmonic wave has

the form:

$$\sigma_{\theta\theta}^* = \left\{ (\bar{\alpha}a) H_{n-1}(\bar{\alpha}a) \left[(n^2 - 1) \bar{\beta}a H_{n-1}(\bar{\beta}a) - (n^3 - n + \frac{1}{2} \bar{\beta}^{-2} a^2) H_n(\bar{\beta}a) \right] - \right.$$

$$\left. - H_n^{(1)}(\bar{\alpha}a) \left[(n^3 - n + \frac{1}{2} \bar{\beta}^{-2} a^2) \bar{\beta}a H_{n-1}(\bar{\beta}a) - n^2 + n - \frac{1}{4} \bar{\beta}^2 a^2 \right] \right\}$$

The stress concentration under the influence of a shear wave (or transverse waves) has the following form

$$\sigma_{\theta\theta}^* = \frac{\sigma_{\theta\theta}}{\sigma_0} \Big|_{r=a} = \frac{y}{\pi} \left(1 - \frac{1}{\chi^2}\right) \sum_{n=0}^{\infty} \epsilon_n i^{n+1} S_n^* \cos n\theta e^{i\omega t}$$

Where

$$S_n^* = \left\{ (n^2 - 1) \bar{\beta}a H_{n-1}(\bar{\beta}a) - (n^3 - n + \frac{1}{2} \bar{\beta}^{-2} a^2) H_n(\bar{\beta}a) \right\}.$$

$$\sigma_{\theta\theta}^* = -\frac{8}{\pi} \left(1 - \frac{1}{\chi^2}\right) \sum_{m=0}^{\infty} i^{n+1} \frac{n(n^2 - 1 - \frac{\bar{\beta}^2 \alpha^2}{2}) H_n^1(\bar{\alpha}a)}{\Delta_n} \sin n\theta e^{-i\omega t};$$

$$\Delta_n = \bar{\beta}^2 \alpha^2 (n^2 + n - \frac{\bar{\beta}^2 \alpha^2}{2}) H_n^1(\bar{\alpha}a) H_n^{(1)}(\bar{\beta}a) + \bar{\alpha} \bar{\beta} a^2 (n^2 - 1) H_{n-1}(\bar{\beta}a) +$$

$$+ (n - n^2 - \frac{\bar{\beta}^2 \alpha^2}{2}) [\bar{\beta} \alpha H_n(\bar{\alpha}a) H_{n-1}(\bar{\beta}a) + \bar{\alpha} a H_{n-1}(\bar{\alpha}a) H_n(\bar{\beta}a)]$$

Stress at the boundary of a rigid inclusion under the influence of a shear wave ($r = a$). in dimensionless form has the form:

$$\sigma_{rr}^* = \frac{4}{\pi} \left\{ -\frac{(1-\eta)H_1^{(1)}(\bar{\alpha}a)}{\delta_1} \sin \theta + \sum_{n=2}^{\infty} \frac{i^{n+1}H_n(\bar{\alpha}a)}{\Delta n} \sin n\theta \right\} e^{-i\omega t}$$

$$\sigma_{r\theta}^* = \frac{2}{\pi} \left\{ -\frac{i\bar{\beta}a^2}{\bar{\beta}^3 a^3 H_1^{(1)}(\bar{\beta}a) + 8\eta \left(\frac{\bar{\beta}^2 a^2}{2} H^{(1)}_0(\bar{\beta}a) - \bar{\beta}a H_1(\bar{\beta}a) \right)} - \right.$$

$$-\frac{2}{\delta_1} \left[(1+\eta)H_1(\bar{\alpha}a) - \bar{\alpha}a H_0(\bar{\alpha}a) \cos \theta \right] -$$

$$\left. -2 \sum_{n=2}^{\infty} \frac{i^{n+1} \left[-nH_n^{(1)}(\bar{\alpha}a) + (\bar{\alpha}a)H_{n-1}(\bar{\alpha}a) \right]}{\Delta n} \cos n\theta \right\} e^{i\omega t}$$

$$\delta_1 = -4\eta H_1^{(1)}(\bar{\alpha}a)H_1(\bar{\beta}a) + (1+\eta)\bar{\alpha}a H_0(\bar{\alpha}a) + H_0(\bar{\beta}a),$$

$$\Delta n = n\bar{\alpha}a H_{n-1}(\bar{\alpha}a)H_{n1}(\bar{\beta}a) + n(\bar{\beta}a)H_{n-1}(\bar{\beta}a)H_n(\bar{\alpha}a) -$$

$$-\alpha\bar{\beta}a^2 H_{n-1}^{(1)}(\bar{\alpha}a)H_{n-1}^{(1)}(\bar{\beta}a).$$

Here $\eta = \rho/\rho_1$ is the ratio of the ambient density to the inclusion density. Under the influence of longitudinal waves in a rigid inclusion, the stress tensor components σ_{rr}^* and $\sigma_{r\theta}^*$ take the form:

$$\sigma_{rr}^* = -\frac{2}{\pi} \left\{ i \left[(\bar{\alpha})H_1^{(1)}(\bar{\alpha}a) \right]^{-1} - 2 \left[(1+\eta)H_1^{(1)}(1-\eta)H_1^{(1)}(\bar{\beta}a) - \bar{\beta}a H_0(\bar{\beta}a) \right] \cos(\theta / \Delta_1) \right.$$

$$\left. + 2 \sum_{n=2}^{\infty} i^{n+1} \left[-\bar{\beta}a H_0(\bar{\beta}a) / \Delta_{r1} \right] \cos n\theta \right\} e^{-i\omega t};$$

$$\sigma_{r\theta}^* = -\left(\frac{2}{\pi} \right) \left\{ 2(1-\eta) - H_1^{(1)}(\bar{\beta} \sin \theta / \Delta_1) + 2 \sum_{n=2}^{\infty} i^{n-1} \left[nH_n^{(1)}(\bar{\beta}a) / \Delta_n \right] \sin \theta \right\} e^{-i\omega t}$$

$$\sigma_{\theta\theta}^* = (1 - \frac{1}{\chi^2}) \sigma_{rr}^*; \chi^2 = \frac{\bar{c}_\alpha^2}{\bar{c}_\beta^2}.$$

The stress tensor components σ_{rr}^* and $\sigma_{\theta\theta}^*$ of an elastic cylindrical inclusion at $r = a$ under the action of longitudinal waves take the following form:

$$\sigma_{rr}^* = -\frac{\eta}{\pi} \sum_{m=0}^{\infty} \frac{\epsilon_n i^{n+1} I_n(\bar{\alpha}_2 a) \bar{\beta}_1^2 a^2}{\Delta_n} \times \left\{ \beta_1 a H_{n-1}^{(1)}(\bar{\beta}_1 a) - (n^2 + n - \frac{\bar{\beta}_1^2 a^2}{2} H_n^{(1)}(\bar{\beta}_1 a) \right\} \cos n e^{-i\omega t} \beta;$$

$$\sigma_{r\theta}^* = -\frac{\eta}{\pi} \sum_{n=0}^{\infty} \frac{\epsilon_n i^{n+1}}{\Delta n} \left\{ \left[\left(\frac{1}{\bar{\chi}^2} - 1 \right) \left(\frac{n \bar{\beta}_1^2 a^2}{2} (n-1) - n^2 (n^2 - 1) - \eta \frac{\bar{\beta}_1^4 a^4}{4} \right) + \right. \right. \\ \left. \left. + \frac{\bar{\beta}_1^2 a^2}{4} \eta (n(n+1) - \frac{1}{2} \bar{\beta}_1^2 a^2) \right] * \right. \\ \left. * I_n(\alpha_2 a) + \left(\frac{1}{\bar{\chi}^2} - 1 \right) \left(n^3 - n + \frac{1}{2} \bar{\beta}_1^2 a^2 \right) (\alpha_2 a) I_{n-1}(\alpha_2 a) H_n^{(1)}(\bar{\beta}_1 a) + \right. \\ \left. + \left(\frac{1}{\bar{\chi}^2} - 1 \right) (1 - n^2) (\alpha_2 \bar{\beta} a^2) I_{n-1}(\alpha_2 a) H_{n-1}(\bar{\beta}_1 a) \right\} \cos n \theta e^{-i \omega t}, \\ \text{где } \eta = \rho_2 / \rho_1 \quad \bar{\chi}^2 = \frac{\bar{\beta}_1^2}{\alpha_1^2}.$$

The Bessel and Hankel functions of the real argument are tabulated in the same way as the trigonometric functions of the logarithm and are known functions of their argument. Since these are real functions, they describe steady waves, in fact, a steady cylindrical wave is a superposition of two traveling waves: one diverges from the axis of the cylinder, and the other converges with it. Since at $r=0$ the phases of these two waves are opposite, they cancel each other, and therefore the amplitude of the steady wave remains finite at $r=0$ [1]. This statement is used in solving the problem of wave propagation in viscoelastic media. Cylindrical wave $I_0(kr)$ corresponds to a plane

steady wave $\cos(kx-\pi/4)$; the amplitudes of successive maxima, due to the distribution of energy on ever larger cylindrical surfaces, are not constant, but decrease with distance. Cylindrical wave $N_0(kr)$ asymptotically corresponds to a plane wave $\sin(kx-\pi/4)$. On the axis of the cylinder ($r=0$) $\text{grad } u$ of the incident and reflected waves are the same, and the amplitude of the reflected wave becomes infinitely large, in contrast to the case of a plane wave; therefore, the function N_0 on the axis of the cylinder ($r=0$) has a pole, i.e. represents the source.

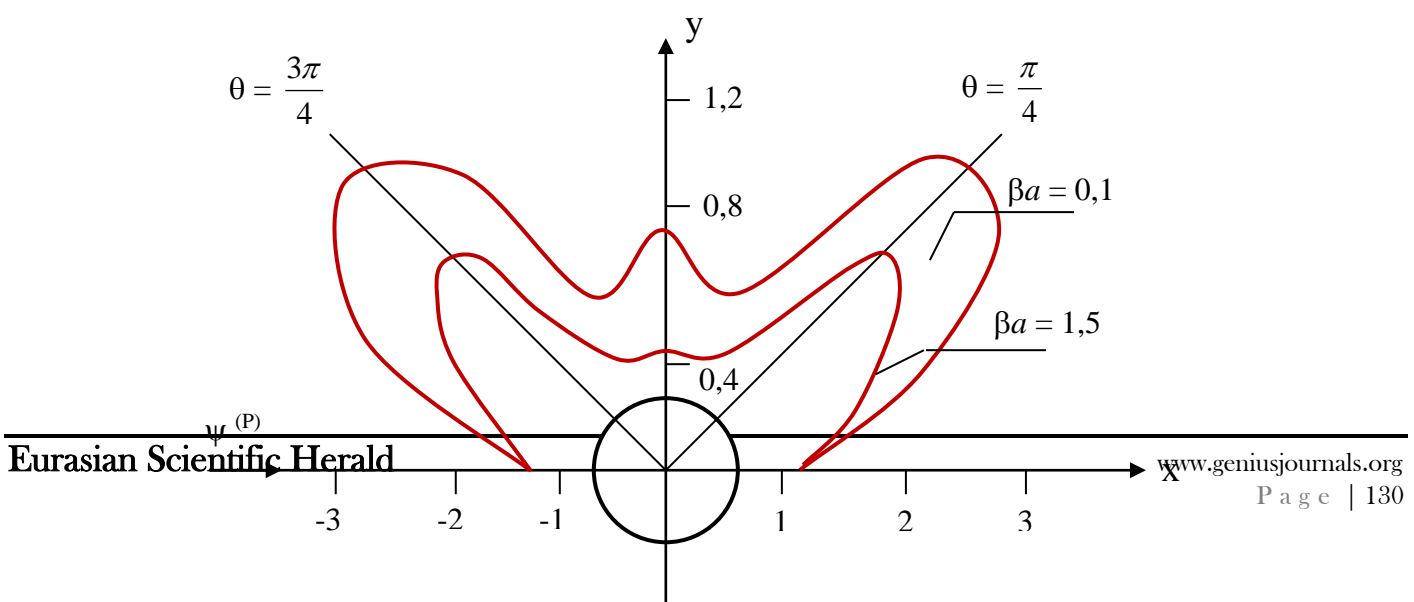


Fig.2. Distributions $|\sigma_{\theta\theta}|$ at different values of α under the influence of shear waves.

1) $A=0.01$; $\beta = 0.05$; $\alpha = 0.1$; $A=0.05$; $\beta = 0.1$; $\alpha = 0.1$.

and for dimensionless wave numbers in the interval $0.01 \leq \alpha^* \leq 3.0$

The results of distribution calculations for different values of the wave numbers are shown in Fig.2. It should be noted that at and the stress distribution is almost the same as in the static case, while at higher wave numbers the stress distribution differs significantly from the static case. The maximum dynamic stress is 10-15% higher than the static one, and the wave numbers at which the maximum value is reached lie between 0.25-0.75.

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